

Lecture 23

185

$$\phi = \alpha \theta, \quad \underline{\xi} = \hat{\xi}(P) + \alpha \underline{z}$$

$$\frac{d}{d\alpha} \left(\underline{r}' \right) \Big|_{\alpha=0} = \underline{u}' \quad \left(\underline{r}' = \underline{\xi} \underline{d}_3 \right)$$

$$= \frac{d}{d\alpha} \left(\hat{\xi}(P) + \alpha \underline{z} \right) \left[\cos(\alpha \theta) \underline{e}_3 + \sin(\alpha \theta) \underline{e}_1 \right] \Big|_{\alpha=0}$$

$$= \underline{z} \underline{e}_3 + \hat{\xi}(P) \theta \underline{e}_1$$

$$\frac{d}{d\alpha} \left[\hat{N} \left(\hat{\xi}(P) + \alpha \underline{z}, \alpha \theta \right) = -P \cos(\alpha \theta) \right] \Big|_{\alpha=0}$$

$$\Rightarrow \hat{N}_{\underline{z}} \left(\hat{\xi}(P), 0 \right) \underline{z} + \hat{N}_{\theta} \left(\hat{\xi}(P), 0 \right) \theta' = 0$$

(P. 182)

$$\Rightarrow \boxed{\underline{z} = 0} \quad \left(\because \underline{u}' = \hat{\xi}(P) \theta \underline{e}_1 \right)$$

$$\frac{d}{d\alpha} \left[\hat{M}_k \left(\hat{\xi}(P), \alpha \theta \right) \times \theta'' + \hat{\xi}(P) P \sin(\alpha \theta) \right] \Big|_{\alpha=0} = 0$$

no variation

$$\Rightarrow \hat{M}_k \left(\hat{\xi}(P), 0 \right) \theta'' + \hat{\xi}(P) P \theta = 0$$

$$\theta'(0) = \theta'(l) = 0$$

$$\text{or} \quad \theta'' + \frac{\hat{\xi}(P) P}{\hat{M}_k \left(\hat{\xi}(P), 0 \right)} \theta = 0$$

Non-trivial $\theta = \cos \frac{n\pi x}{L} \Leftrightarrow$

$$(*) \quad \frac{\hat{\xi}(P) P}{\hat{M}_k \left(\hat{\xi}(P), 0 \right)} = \frac{n^2 \pi^2}{L^2}, \quad n = 1, 2, \dots \quad \text{Char. eq.}$$

For $n=1$

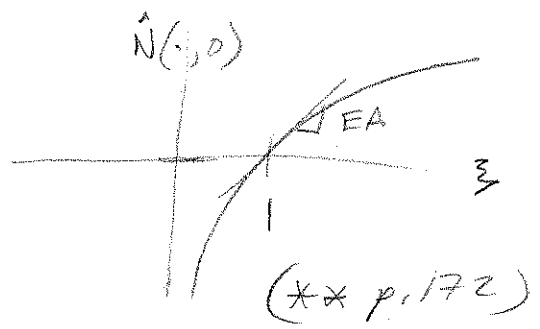
$$g(P) \equiv \frac{P \hat{\xi}(P)}{\hat{M}_K(\hat{\xi}(P), 0)} = \frac{\pi^2}{L^2}$$

Observe: $\hat{\xi}(P), \hat{M}_K(\hat{\xi}(P), 0) > 0$

"instantaneous bending modulus"

The estimation depends crucially upon the behavior of the function $g(P)$:

Ex) Assume $\hat{N}(\xi, 0) \equiv EA \ln \xi$



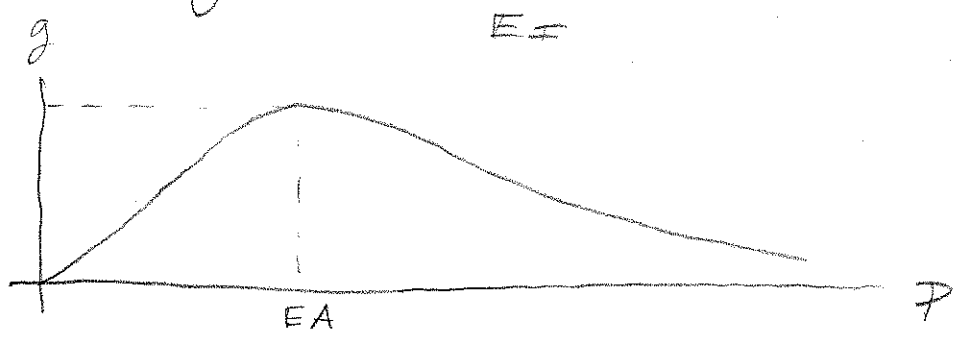
$$\Rightarrow EA \ln \xi = -P$$

$$\Rightarrow \xi = e^{-P/EA}$$

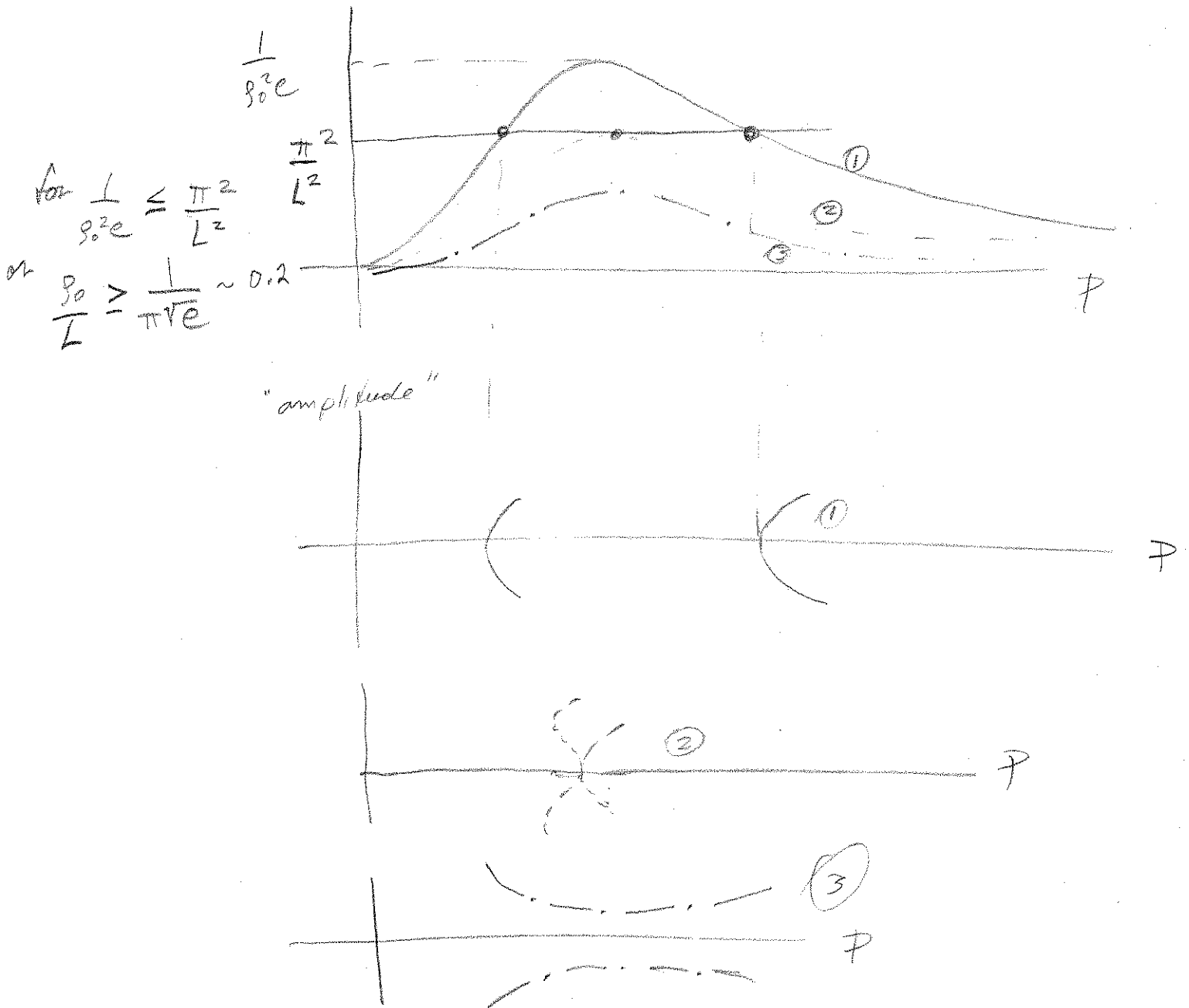
Choose $\hat{M}_K(\hat{\xi}(P), 0) \equiv EI$ (const. "bending" modulus)

Then $g(P) = \frac{P e^{-P/EA}}{EI}$

$\frac{A}{I} e \equiv \frac{1}{\rho_0^2 e}$
rod gyration

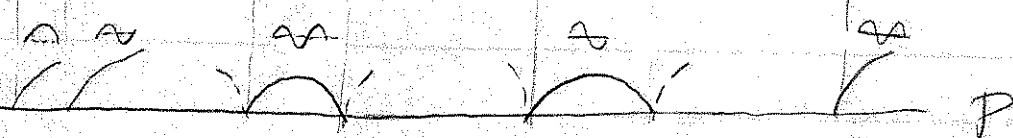
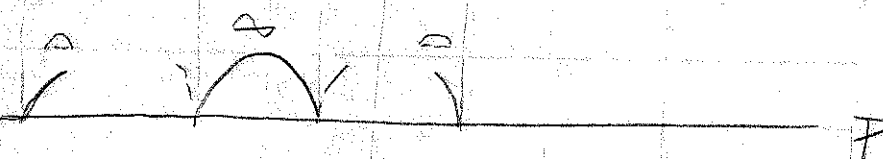
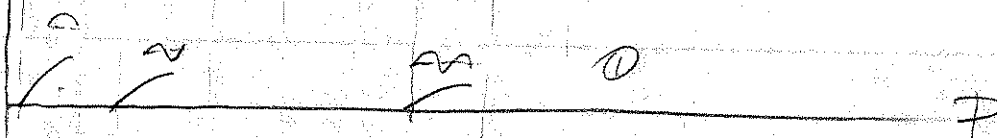
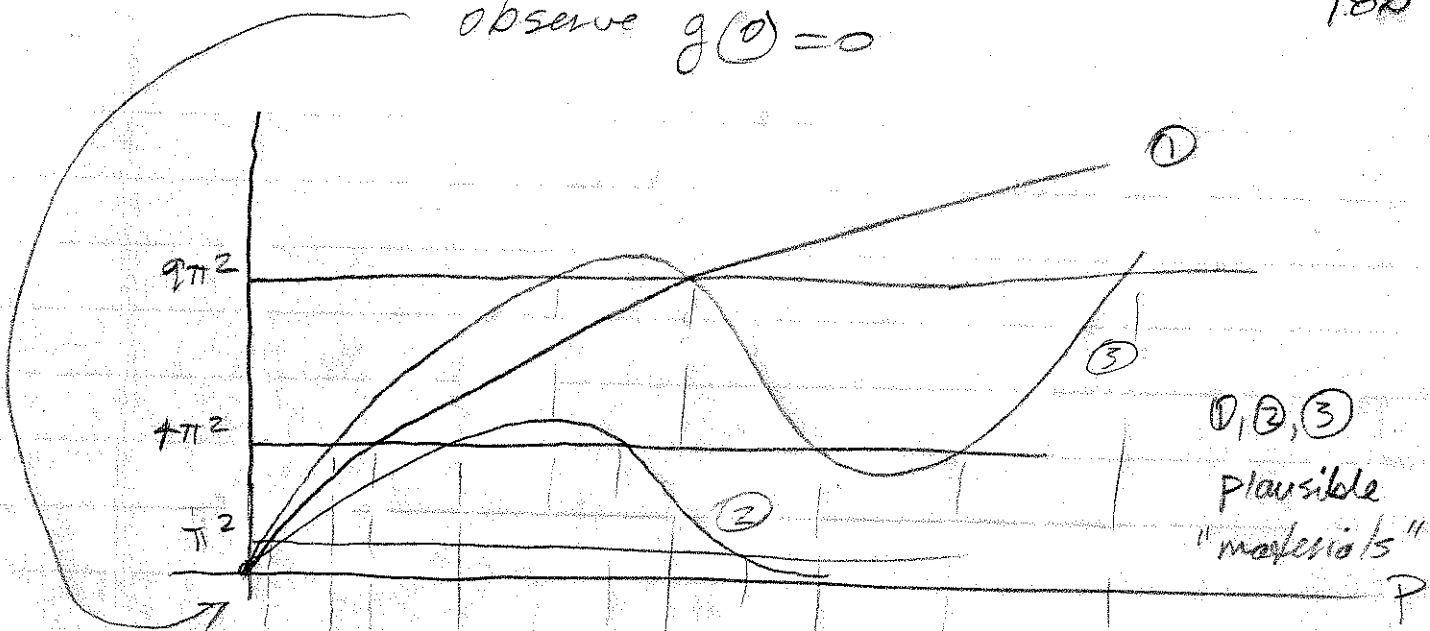


To "solve" the char. eq. (*) p.173, we compare the graphs



Of course for other materials (cross-sections) other exotic bit. diagrams are possible:

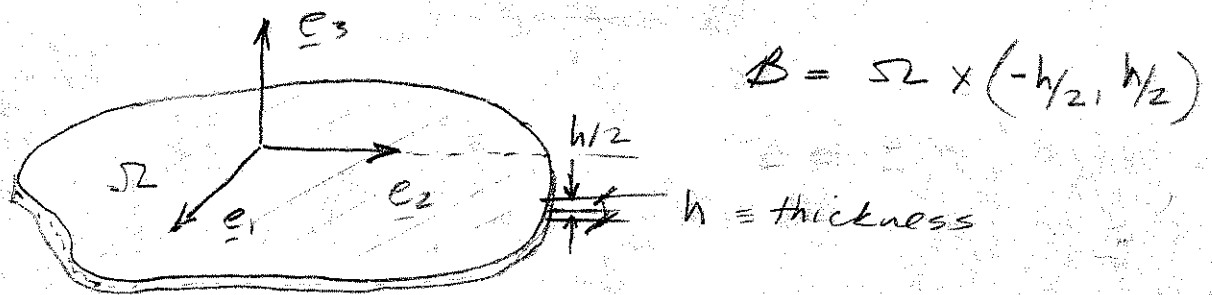
observe $g(0) = 0$



Special Cosserot Theory of Plates (shells)

Roughly speaking, a plate (shell) is a body for which one dimension — called the thickness — is much smaller than the other two dimensions. As with rods, we will freely bounce back and forth between the direct approach and the constrained (3-d) approach.

For simplicity, we shall assume that the reference configuration is flat with the "middle surface" contained in the plane spanned by \underline{e}_1 & \underline{e}_2 :



Let $\underline{s} = X_1 \underline{e}_1$ denote the position vector of a material point in Ω . The deformed middle surface (at time t) is specified by

$$\underline{s} \mapsto \underline{r}(\underline{s}, t) \equiv \tilde{\underline{r}}(X_1, X_2, t),$$

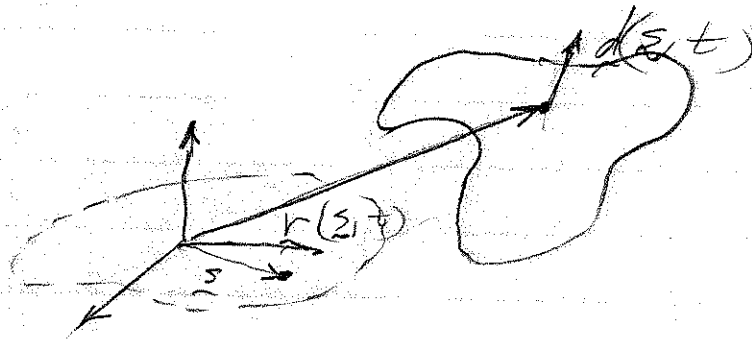
where $\underline{r} : \Omega \in \mathbb{E}^2 \rightarrow \mathbb{E}^3$. To model the effects of thickness, we attach a simuli

this is what is typically called a plate = flat shell

director field $\underline{\tilde{d}}(\underline{s}, t)$ to each material point, which represents the orientation of the "through-thickness fiber" original aligned with \underline{e}_3 in the reference configuration.

Constrained 3-d motion

$$\underline{f}_c(\underline{X}, t) = \underline{r}(\underline{s}, t) + \underline{X}_3 \underline{\tilde{d}}(\underline{s}, t)$$



Then

$$\underline{F}_c(\underline{X}, t) = \underline{\nabla} \underline{f}_c(\underline{X}, t)$$

$$= \frac{\partial f_i}{\partial X_\alpha} \underline{e}_i \otimes \underline{e}_\alpha + \frac{\partial f_i}{\partial X_3} \underline{e}_i \otimes \underline{e}_3$$

$$= \left(\frac{\partial \tilde{r}_i}{\partial X_\alpha} + X_3 \frac{\partial \tilde{d}_i}{\partial X_\alpha} \right) \underline{e}_i \otimes \underline{e}_\alpha$$

$$+ \underline{d} \otimes \underline{e}_3$$

$$= \left(\tilde{r}_{i,\alpha} + X_3 \tilde{d}_{i,\alpha} \right) \underline{e}_i \otimes \underline{e}_\alpha + \underline{d} \otimes \underline{e}_3$$

Aside $\underline{s} \mapsto \underline{r}$ is differentiable
(at \underline{s}) if

$$\underline{r}(\underline{s} + \alpha \underline{\xi}) = \underline{r}(\underline{s}) + \nabla_{\underline{r}}(\underline{s}) \underline{\xi} + o(|\underline{\xi}|)$$

as $|\underline{\xi}| \rightarrow 0$

where $\nabla_{\underline{r}}(\underline{s}) \in L(\mathbb{E}^2, \mathbb{E}^3)$ (for fixed \underline{s}).

Choose $\underline{\xi} = \underline{e}_2$. Then

$$\frac{d}{d\alpha} \underline{r}(\underline{s} + \alpha \underline{e}_2) \Big|_{\alpha=0} = \frac{d}{d\alpha} \tilde{\underline{r}}(\underline{x}_1, \underline{x}_2 + \alpha) \Big|_{\alpha=0}$$

$$\Rightarrow \nabla_{\underline{r}}(\underline{s}) \underline{e}_2 = \frac{d\tilde{\underline{r}}}{d\underline{x}_2}(\underline{x}_1, \underline{x}_2)$$

$$\underline{e}_i \cdot \nabla_{\underline{r}}(\underline{s}) \underline{e}_2 = \frac{d\tilde{r}_i}{d\underline{x}_2}(\underline{x}_1, \underline{x}_2)$$

$$\Rightarrow \underline{e}_i \cdot \nabla_{\underline{r}}(\underline{s}) \underline{e}_\alpha = \frac{d\tilde{r}_i}{d\underline{x}_\alpha}(\underline{x}_1, \underline{x}_2)$$

$$(*) \quad \text{and} \quad \nabla_{\underline{r}}(\underline{s}) = \frac{d\tilde{r}_i}{d\underline{x}_\alpha}(\underline{x}_1, \underline{x}_2) \underline{e}_i \otimes \underline{e}_\alpha$$

(likewise for $\nabla_{\underline{d}}$)

Returning to (*) p. 188, we have

$$\underline{F}_c = \nabla_{\underline{r}} + \underline{x}_3 \nabla_{\underline{d}} + \underline{d} \otimes \underline{e}_3$$

Note: $\nabla_{\underline{r}}(\underline{s})$ and $\nabla_{\underline{d}}(\underline{s})$ are coordinate-free.